

TITLE

Methods and apparatus for starting up emission-free
gas-turbine power stations

5

TECHNICAL FIELD

The present invention relates to a power generation
plant having at least one gas turbine cycle with heat-
10 recovery boiler and at least one steam turbine cycle
operated via the heat-recovery boiler, the gas turbine
cycle being designed to be semi-closed and essentially
free of emissions and essentially comprising a
compressor, a combustion chamber arranged downstream of
15 the compressor, a gas turbine arranged downstream of
the combustion chamber, a heat-recovery boiler arranged
downstream of the gas turbine, and at least one
generator coupled to the gas turbine. The invention
also relates to methods of starting up and of operating
20 such a power generation plant.

PRIOR ART

Within the limits of general attempts to develop power
25 stations which produce as little environmental
pollution as possible, there are a large number of
different projects whose aim is to develop emission-
free gas turbine power stations having a semi-closed
CO₂/H₂O cycle. In this case, the natural gas used as
30 fuel is burned with as far as possible pure oxygen.
Combustion gases which consist virtually only of carbon
dioxide and water are produced under these
circumstances. If water is condensed out of the working
medium, largely pure carbon dioxide is obtained, which
35 can be liquefied by compression and can be used and
disposed of in different ways.

To utilize the high temperatures at the turbine outlet,
a steam generator is used as a rule, the steam produced

being used to drive a bottoming steam turbine. Since the turbine outlet temperature under normal pressure conditions for CO₂/H₂O mixtures is higher than in conventional gas turbines, the steam cycle in such systems delivers up to about 50% of the total output.

Alternatively, the generated steam can be pre-expanded in a topping steam turbine in order to then be mixed with the working medium of the gas turbine upstream of, in or downstream of the combustion chamber. The injected steam, after flowing through the heat-recovery boiler, can then be condensed out together with the water produced by the combustion. Both concepts are described in more detail in EP 0 731 255 B1.

The use of emission-free gas-turbine power stations is nowadays considered in particular in the oil and gas industry, since the separated carbon dioxide can be used there to a considerable extent (Enhanced Oil Recovery, EOR) and, in part, already heavy taxes have to be paid for emitted carbon dioxide. In the oil and gas industry, however, power stations are often operated in an environment in which it is difficult or not possible to draw start-up power from the network (remote coastal locations, drilling platforms, etc.). This problem is made more difficult in emission-free power stations of the type described above by virtue of the fact that an air separation plant mainly of cryogenic design has to be started up before the start-up of the turbine, this air separation plant, for a period of 2 to 4 hours, requiring approximately 10% of the network output of the power station in order to achieve a stable operating point.

For an autonomous start-up operation, current which is generated by the generators of the integrated steam turbines can be used in conventional gas-turbine power stations. As an example of this, an arrangement according to US Patent 5,148,668 can be used, in which

a hot-water reservoir charged during operation delivers the steam required for the start-up. In order to provide the requisite steam for a longer period, auxiliary firing of the hot-water reservoir is provided
5 in this patent. Because no rapid start is possible for emission-free power stations, recourse cannot be had to the concept of the use of a hot-water reservoir.

DESCRIPTION OF THE INVENTION

10

One object of the invention is therefore to provide an emission-free, semi-closed power station plant of the abovementioned type, that is to say according to the preamble of patent claim 1, which power station plant
15 operation and start-up with minimum start-up output.

The present invention achieves this object by first means being arranged which alternatively or additionally allow hot gas to be fed into the hot-gas
20 path between gas turbine and heat-recovery boiler, and by second means being arranged which alternatively or additionally allow exhaust gas to be expelled from the exhaust-gas path downstream of the heat-recovery boiler.

25

This surprisingly simple modification of the gas turbine cycle allows the heating of the heat-recovery boiler while the turboset is stopped or still does not have sufficiently high capacity (or even no longer has
30 sufficiently high capacity), in such a way that the steam turbine cycle can be operated in an energy-generating or in particular current-generating manner. In other words, the heat-recovery boiler through which an exhaust-gas mixture of the gas turbine plant flows
35 during normal operation is operated as an auxiliary-fired steam generator. The generators of the steam turbines to which the steam thus generated is admitted, given an appropriate design of the auxiliary firing, generate sufficient current in order to be able to

start up both an air separation plant, possibly present for the supply with pure oxygen, and the gas turbine. In addition, the modification allows the steam turbine cycle to be operated in a current-generating manner on its own, and the plant can therefore also assume the function of an emergency generating unit, which may become necessary, for example, during possible outage times of air separation plant and/or gas turbine. In this case, the exhaust gas expelled from the exhaust-gas path is normally discharged via an auxiliary stack.

According to a first, especially simple and preferred embodiment of the invention, the first and second means are switch-over members which allow the feeding-in or expelling in particular by resetting air flaps.

According to a further embodiment of the invention, the additional hot gas, to be alternatively or additionally fed into the hot-gas path, is provided by one or more auxiliary burners which are preferably supplied with fresh air via a blower. In principle, however, it is also possible to provide the hot gas in another way, for example via heat exchangers, catalysts, etc.

The power generation plant according to the invention is advantageously operated as a $\text{CO}_2/\text{H}_2\text{O}$ plant, that is to say a $\text{CO}_2/\text{H}_2\text{O}$ gas turbine cycle is involved in which CO_2 and H_2O produced, via corresponding means for compression and/or means for cooling, are removed from the gas turbine cycle, in particular preferably in such a way as to branch off directly downstream of the compressor, and in particular in a solid and/or liquid form, the gas turbine cycle being supplied with largely pure oxygen in particular via an air separation plant. In this case, the air separation plant may be of cryogenic design or may be based on a diaphragm process.

According to another preferred embodiment of the invention, the steam turbine cycle is of essentially closed design and has at least one steam turbine and at least one generator coupled thereto. In this case, the steam turbine cycle, with the use solely of hot gas fed in via the first means, while exhaust gases are simultaneously discharged via the second means, can be operated in such a way that the generator generates sufficient energy in order to put the gas turbine plant and an air separation plant possibly present into operation, or respectively in order to serve as emergency generating unit in the event of a failure of the gas turbine plant. In addition, in order to meet the special requirements during start-up or during operation as emergency generating unit, a further switch-over member, via which ambient air can be drawn in, can preferably be arranged upstream of the compressor.

Depending on requirements, the steam turbine arranged in the steam turbine cycle may be designed as a bottoming steam turbine or as a topping steam turbine, the partly expanded exhaust steam of which, after injection into the cycle medium upstream of, in and/or downstream of the combustion chamber, is expanded to ambient pressure in the gas turbine, with power being delivered, in particular a switch-over member being provided with which the exhaust steam can be directed past the gas turbine directly for liquefaction into a cooler arranged in the gas turbine cycle.

Further preferred embodiments of the power station plant according to the invention are described in the dependent patent claims.

The present invention also relates to a method of starting up a power generation plant as described above, which is characterized in that, first of all, in a first phase, the steam turbine cycle is put into

operation with hot gas fed in via the first means, while at the same time the exhaust gases are at least partly expelled via the second means, then, in a second phase, the generator is motor-driven with current by a
5 generator arranged in the steam turbine cycle in order to start up the turboset, the compressor, via an air flap arranged upstream and/or via the second means opened in both directions, drawing in fresh air or a combustion-gas mixture and delivering it through the
10 combustion chamber, in which, possibly with additional feeding of largely pure oxygen, fuel is fired, so that the turbine starts to assist the motor-driven generator and finally serves as sole drive, the hot exhaust gases of the gas turbine progressively taking over the steam
15 generation in the heat-recovery boiler and completely taking over the steam generation in the heat-recovery boiler at the end. In this case, the separation into individual phases is not to be seen in an absolutely strict sense; corresponding optimum control of the
20 start-up process with partly overlapping sections can be determined by the person skilled in the art.

Furthermore, the present invention relates to a method of starting up a power generation plant as described
25 above, which is characterized in that, first of all, in a first phase, the steam turbine cycle is put into operation with hot gas fed in via the first means, while at the same time the exhaust gases are at least partly expelled via the second means, in that, after
30 the turboset, operated with air as substitute medium via an air flap arranged upstream of the compressor, is running in a self-sustaining manner, in a second phase, the gas turbine cycle is closed via the first and second means and the air flap, and largely pure oxygen
35 is fed as an oxidizing agent to the combustion chamber, gas being continuously expelled from the cycle in order to compensate for the feed of oxygen and fuel, and the composition of the circulating gas progressively approaching an equilibrium, in which the separation and

liquefaction of the combustion products can be started. The equilibrium is in this case achieved when the combustion-gas mixture essentially comprises only CO₂ and H₂O, and nitrogen, oxygen or the like which could
5 disturb the condensation process of the CO₂ are no longer present. In this case, the current available after the first phase via the generator can at least partly be used for operating the air separation plant and thus for providing largely pure oxygen for the
10 combustion process in the combustion chamber.

In addition, the power station plant according to the invention may be run in such a way that, when the gas turbine cycle is not operating, only the steam turbine
15 cycle is operated via the feeding-in of hot air with the first means and via the expelling of exhaust gases with the second means, and that the generator arranged in the steam turbine cycle thus provides current in particular in the sense of an emergency generating
20 unit.

Further preferred embodiments of the methods according to the invention are described in the dependent patent
25 claims.

25
BRIEF DESCRIPTION OF THE FIGURES

The invention is to be explained in more detail below with reference to exemplary embodiments in connection
30 with the figures, in which:

- Fig. 1 shows a scheme of an emission-free gas-turbine power station according to the prior art;
- 35 Fig. 2 shows a scheme of an emission-free gas-turbine power station according to the invention with bottoming steam turbine; and

Fig. 3 shows a scheme of an emission-free gas-turbine power station according to the invention with topping steam turbine.

5 WAYS OF IMPLEMENTING THE INVENTION

Fig. 1 shows the scheme of an emission-free power station with $\text{CO}_2/\text{H}_2\text{O}$ gas turbine and downstream steam cycle with bottoming steam turbine according to the prior art. A $\text{CO}_2/\text{H}_2\text{O}$ turboset, consisting of a compressor 1, a combustion chamber 2, a turbine 3 and a generator 8 arranged on a common shaft 22, is interconnected to form a closed cycle via a heat-recovery boiler 4 and a cooler 5 serving as heat sink. The hot gases issuing from the gas turbine 3 are fed to the heat-recovery boiler via the hot-gas path 23, and the exhaust gases cooled in the heat-recovery boiler 4 are fed downstream of the heat-recovery boiler 4 via the exhaust-gas path 40 to the condenser 5. Up to a limit predetermined by the cooling-water temperature, any desired proportion of the water contained in the working medium can be condensed out by means of the cooler 5. The carbon dioxide produced by the combustion of, for example, natural gas is branched off in steady-state operation by a compressor 6, brought to the pressure required for further use, dried further and liquefied in the cooler 7 and removed from the process via the line 32. In practice, this compression process is advantageously carried out in several stages with interim cooling and drying. Technically pure oxygen, which is obtained in an air separation plant 9 (not described further here and only shown schematically), is used for the oxidation of the fuel in the combustion chamber 2.

35

The steam obtained in the heat-recovery boiler, within the limits of a conventional cycle arrangement, is admitted to a bottoming steam turbine 10 with generator 11. In this case, the steam cycle comprises the

bottoming steam turbine 10, a condenser 30 downstream of it and a pump 31 downstream of said condenser 30, the pump 31 feeding the condensate to a feedwater tank/deaerator 24. The feedwater is fed downstream of
5 the feedwater tank 24 via a pump to an economizer 26 arranged in the heat-recovery boiler 4 and then to the steam drum 27. The steam drum 27 is connected to an evaporator 28, which is likewise arranged in the heat-recovery boiler, and the steam produced in the steam
10 drum 27 is normally superheated in a superheater stage 29 and then fed to the steam turbine 10.

In order to be able to now start up this system in a largely autonomous manner, the plant is equipped with
15 the additional components shown in fig. 2. By means of air flaps or a differently realized switch-over member 12 arranged in the hot-gas path 23, the heat-recovery boiler 4 is switched over on the inlet side from the turbine outlet to one or more auxiliary burners 13
20 which are supplied with air from one or more blowers 14. On the outlet side, the heat-recovery boiler is connected to an auxiliary stack 16 by a further switch-over member 15 arranged in the exhaust-gas path 40. The flue gases produced in the combustion chamber 13 can
25 escape via this stack. In this way, steam can be generated in the heat-recovery boiler before the gas turbine plant 1 - 3 is put into operation. The bottoming steam turbine 10 can now generate via its generator 11 the current which is required in order to
30 operate the air separation plant 9 and start up the gas turbine 1 - 3.

To start up the gas turbine, with switch-over members 12 and 15 opened on both sides (i.e. gas can flow both
35 from 3 and from 13 via 12 in the direction of 4, or from 4 via 15 in the direction of both 16 and 5), the generator 8 is motor-driven and the burner 2 is put into operation with fuel and oxygen from the air separation plant 9. The output of the auxiliary burners

13 and blower(s) 14 is continuously reduced until the exhaust gases of the gas turbine have reached a sufficiently high temperature. The cycle is then closed by means of the switch-over members 12 and 15.

5 Alternatively, the plant can be designed in such a way that ambient air is drawn in via a further switch-over member 17 for starting up the gas turbine.

After completion of the start-up phase, the closed
10 cycle first of all contains a typical combustion-gas mixture with high nitrogen and oxygen content. In order to compensate for the inflow of oxygen and fuel, some of the gas located in the cycle is expelled continuously, for example via the auxiliary stack 16.
15 After a short time, the composition of the circulating gas thus approaches stable equilibrium with carbon dioxide and water as the main components, and the plant can be switched over to completely emission-free operation.

20 In addition, such a modified power station plant permits separate operation solely of the steam turbine cycle for the purposes of an emergency generating unit. This may become necessary, for example, if the gas
25 turbine plant has to be shut down on account of a failure of the air separation plant 9, or if the gas turbine plant has to be stopped for other reasons.

Fig. 3 shows a correspondingly equipped plant embodied
30 with a topping steam turbine. This plant additionally contains a switch-over member 18 with which the steam partly expanded in the topping steam turbine is directed past the gas turbine directly for liquefaction into the cooler 5. Alternatively, the partly expanded
35 steam, before the liquefaction, may also be used for preheating the boiler feedwater (obvious to the person skilled in the art and therefore not shown schematically as an additional option in fig. 3). If the gas turbine 1 - 3 is started up, the switch-over

member 18 is brought into its normal operating position, and the steam is expanded to ambient pressure in the gas turbine with power being delivered. Since the topping steam turbine, at the same fuel consumption of the auxiliary burners 13, delivers considerably less power than the bottoming steam turbine according to fig. 2, this construction is suitable for starting up the air separation plant and the gas turbine, but is less suitable for emergency operation.

For the person skilled in the art, it is obvious that the method described above can be applied not only to the two processes described but also to a multiplicity of conceivable process variants which are characterized in that a gas turbine and a steam turbine are combined such that the working medium of the gas turbine is run in an at least partly closed cycle with or without condensing, largely pure oxygen is fed as an oxidizing agent to the cycle, and the steam required is generated in normal operation by utilizing the waste heat of the gas turbine.

LIST OF DESIGNATIONS

| | | |
|----|----|--|
| | 1 | Compressor |
| | 2 | Combustion chamber |
| 5 | 3 | Turbine |
| | 4 | Heat-recovery boiler |
| | 5 | Cooler, condenser |
| | 6 | Compressor |
| | 7 | Cooler |
| 10 | 8 | Generator |
| | 9 | Air separation plant |
| | 10 | Bottoming steam turbine |
| | 11 | Generator |
| | 12 | Switch-over member |
| 15 | 13 | Auxiliary burner |
| | 14 | Blower |
| | 15 | Switch-over member |
| | 16 | Auxiliary stack |
| | 17 | Switch-over member |
| 20 | 18 | Switch-over member |
| | 19 | Topping steam turbine |
| | 20 | Fuel feed |
| | 21 | Oxygen feed |
| | 22 | Shaft |
| 25 | 23 | Line to the heat-recovery boiler, hot-gas path |
| | 24 | Feedwater tank |
| | 25 | Pump |
| | 26 | Economizer |
| | 27 | Steam drum |
| 30 | 28 | Evaporator |
| | 29 | Superheater |
| | 30 | Condenser |
| | 31 | Pump |
| | 32 | Discharge line for carbon dioxide |
| 35 | 33 | Discharge line for water |
| | 34 | Fresh-air feed |
| | 35 | Fuel feed |
| | 36 | Variable hot-gas path |
| | 37 | Variable exhaust-gas path |

- 38 Variable steam path
- 39 Fresh-air feed, fresh air
- 40 Line to the condenser 5, exhaust-gas path